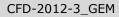




Cooling of GEM detector

CFD-2012-03_GEM 2012/03/13









1. Geometry and **material properties**

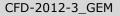


Geometry and material properties EN CFD team

PCB, FR-4, k = 0.25 W m⁻¹ K⁻¹

Copper, $k = 400 \text{ W m}^{-1} \text{ K}^{-1}$

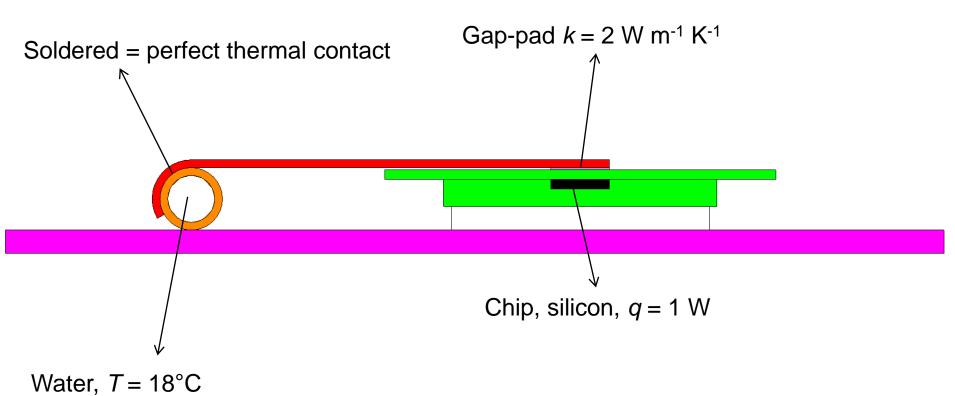
A single chip is considered in the CFD simulations





Geometry and material properties



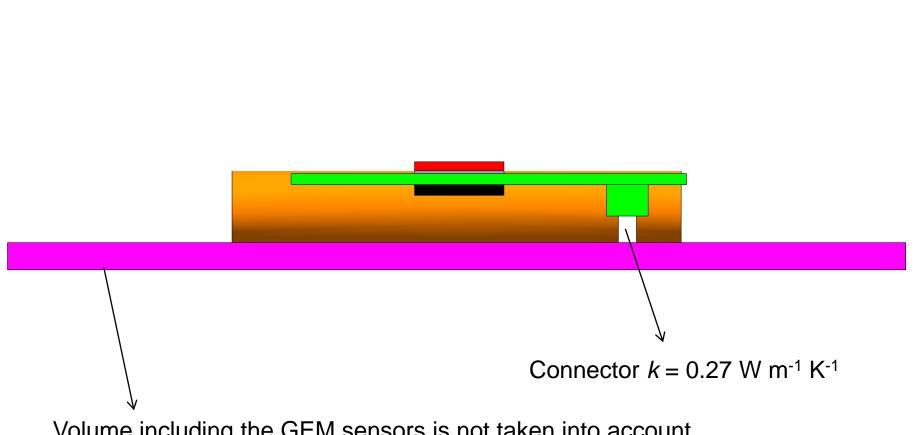


All surfaces are adiabatic but the inner surface of the pipe

CFD team



Geometry and material properties

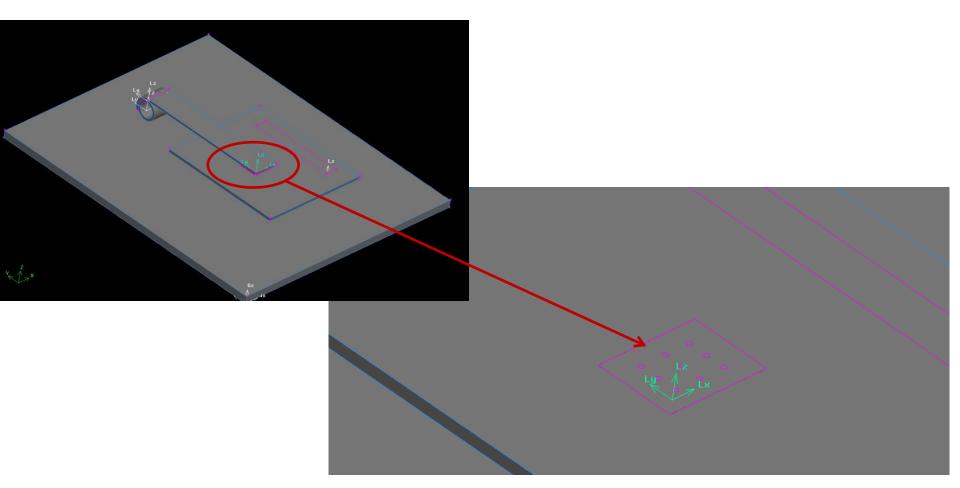


Volume including the GEM sensors is not taken into account. This surface is assumed as adiabatic.

CFD team







9 copper cylinders (0.5 mm diameter) thermally connect the chip to the gap-pad through the PCB

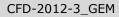
CFD team





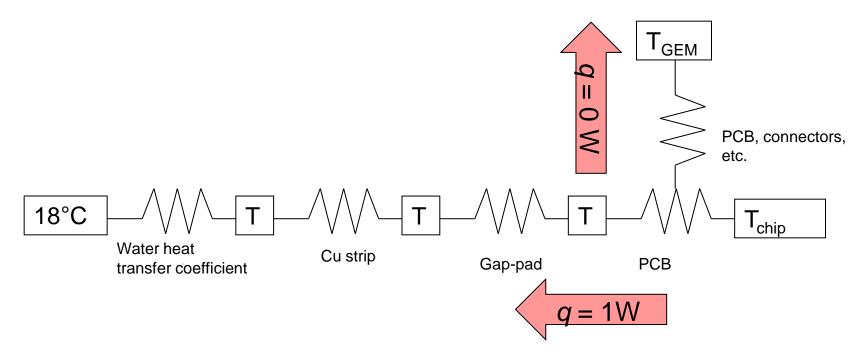
2. Schematic and estimations











□ The temperature of the GEM sensors will be the same as the PCB.

□ In order to reduce the sensor temperature, the thermal resistances must be reduced:

- -) higher water flow rate -> higher heat transfer coefficient
- -) shorter Cu strip or larger "cross section"
- -) thinner gap-pad or higher thermal conductivity
- -) good thermal contact Cu strip/gap-pad and gap-pad/PCB
- -) bigger or higher number of copper thermal vias through the PCB

CFD team





- Heat load per VFAT chip = 1 W
- # VFAT chips = 30
- Total heat load = 1 W * 30 + 4 W = 34W

Di [mm]	Vel. [ms ⁻¹]	Flow rate [kg s ⁻¹]	Temp. rise [K]	Re [-]	HTC [Wm ⁻² K ⁻¹]	Δp (2 m pipe) [bar]
6	1	0.028	0.29	5700	4700	0.06
6	1.8*	0.051	0.16	10200	8200	0.17
4	1	0.013	0.65	3800	4500	0.11
4	1.8*	0.023	0.36	6800	8400	0.28

* Erosion limit for copper pipe

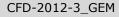
Low water temperature rise is good to achieve a uniform sensor temperature.
The diameter of the pipe may be reduced if needed because of geometrical constraints.







3. Simulation results









Different geometries have been studied and compared:

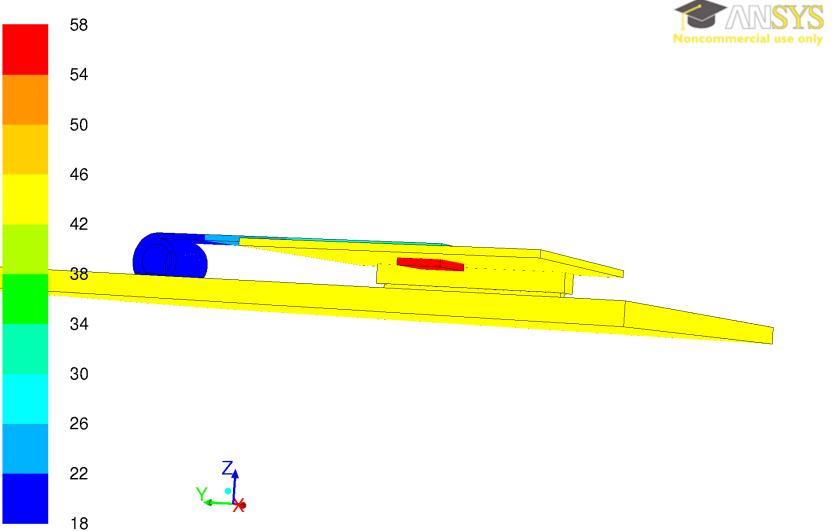
- 1. Reference geometry: 10 mm wide, 1 mm thick copper sheet
- 2. Reference geometry with lower contact area between copper sheet and water pipe
- 3. Y-shaped copper sheet (40 mm at pipe side, 10 mm at PCB side)
- 4. Wider copper sheet (40 mm) with 10 mm wide gap-pad
- 5. Wider copper sheet with wider gap-pad (40 mm)





1) Reference geometry

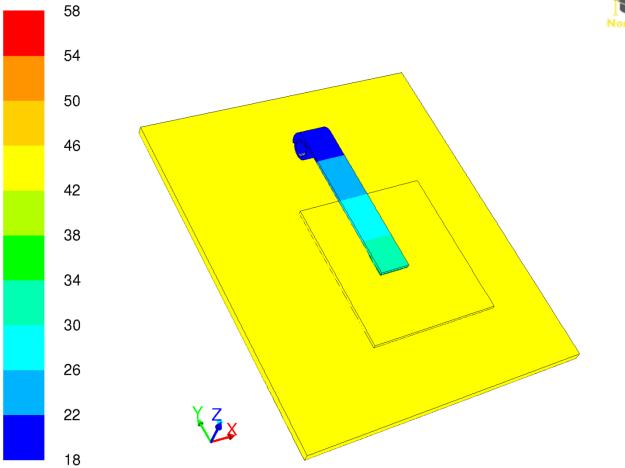






1) Reference geometry

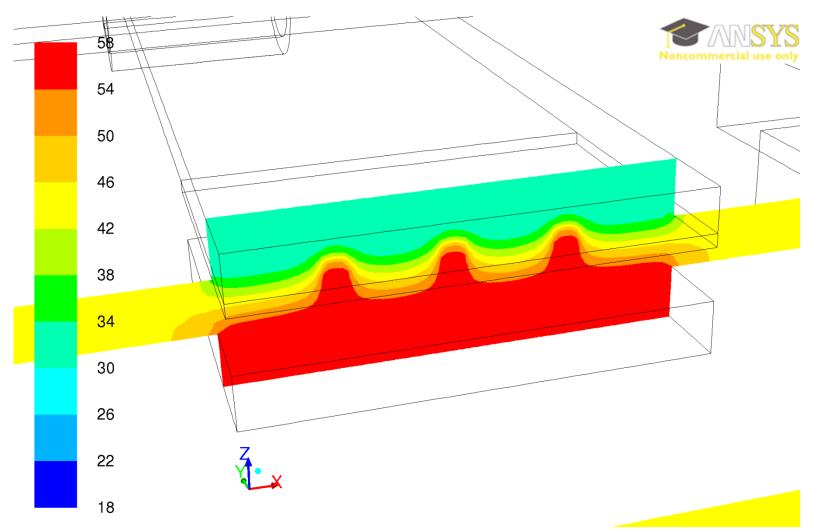








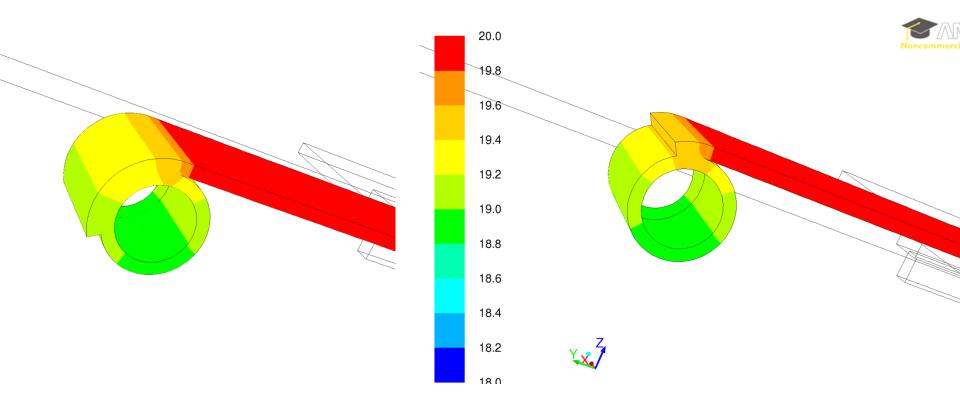
1) Reference geometry



Cross section with copper thermal vias

CFD team 2) Smaller copper sheet/pipe contact area

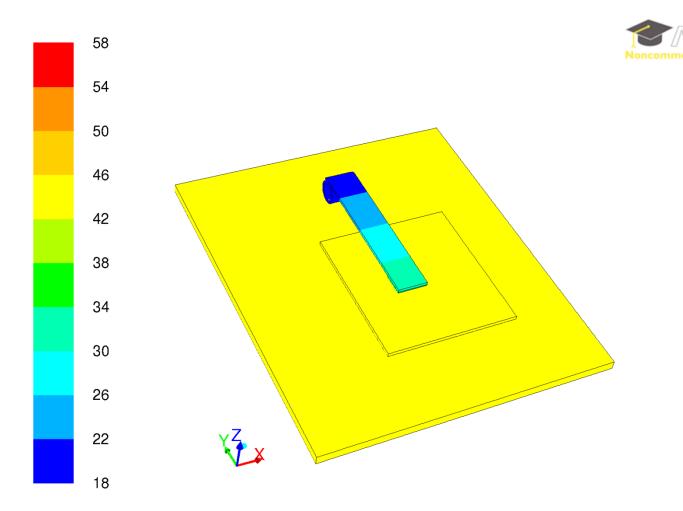




□ Negligible influence of the reduced soldering area

□ Around 1~2 K temperature drop due to the water heat transfer coefficient

CFD team 2) Smaller copper sheet/pipe contact area



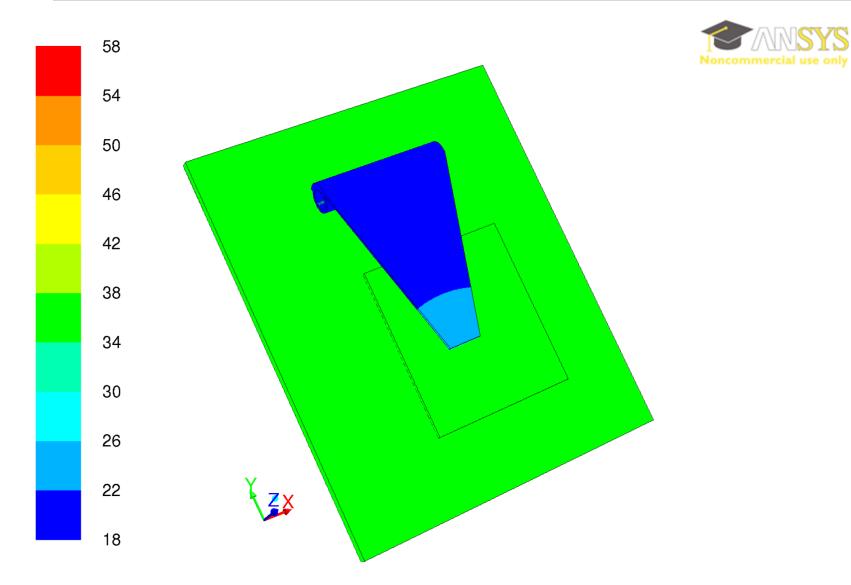
Same thermal performance as the reference geometry





3) Y-shape copper sheet

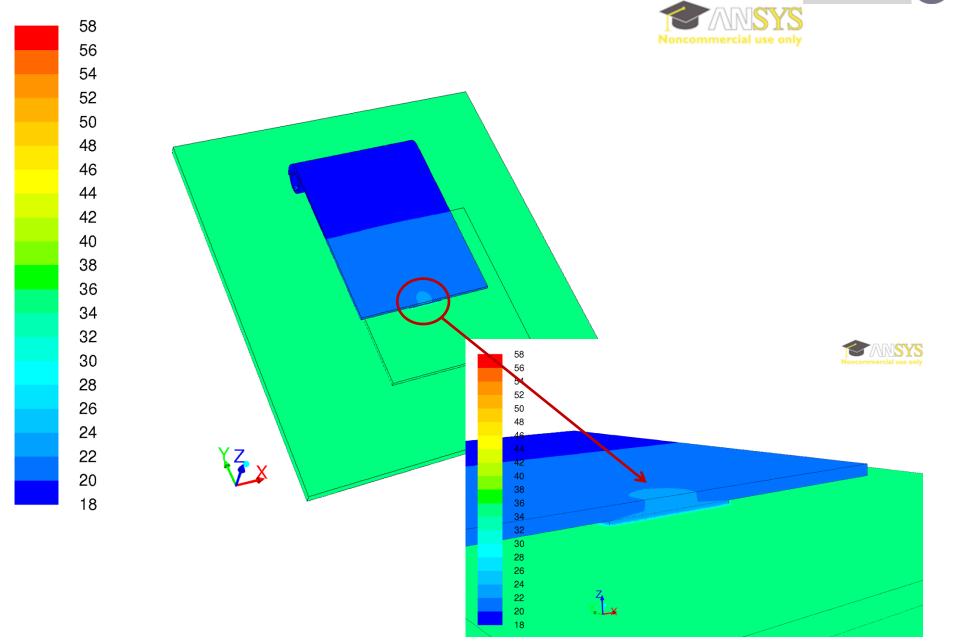






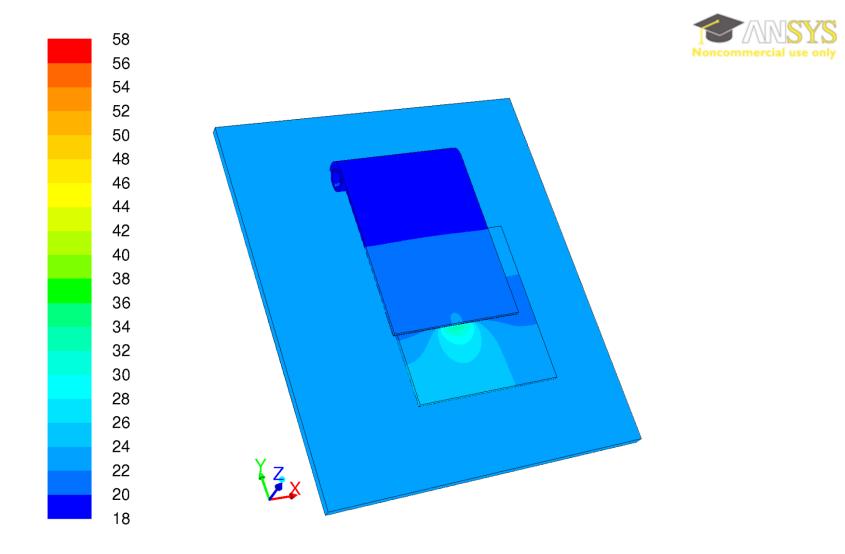
CERN

CFD team 4) 40 mm wide copper sheet + 10 mm wide gap-pad EN

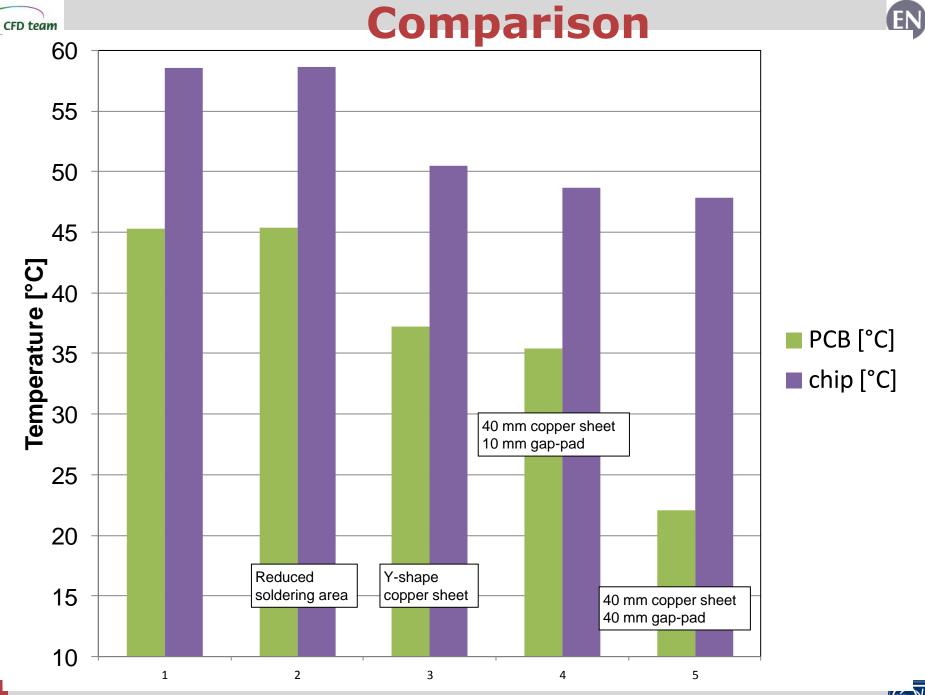


CFD team 5) 40 mm wide copper sheet & gap-pad









CFD-2012-3_GEM





Conclusions

• 6 mm i.d. pipe, with 1 ms⁻¹ water velocity is enough to keep the water temperature rise below 0.3 K and achieve a good enough heat transfer coefficient with low pressure drop.

• Soldering area between copper sheet and pipe does not have a major influence.

• PCB temperature strongly depends on the width of the copper sheet and contact area among copper sheet, gap-pad and PCB.

• With 40 mm wide, 1 mm thick copper sheet + 40 mm X 10 mm gap-pad (0.3 mm thick) the PCB temperature is expected to be below 25 °C.

• Cooling of "dividers" is still an open issue.

• The gas flow around the GEM sensors (not taken into account in the present simulation) helps cooling the sensors and keeping the temperature uniform.

• If possible, increasing the copper sheet thickness is a another "cheap & easy" way to improve the thermal performance.

